

Research Article

# Homotopy Perturbation Method for Analyzing the Effect of Viscous Dissipation on Steady Natural Convection Couette Flow with Convective Boundary Conditions

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## Abstract

This research article presents an analytical study of convective flow in a vertical channel with convective boundary conditions. Because of the nonlinear nature of the governing energy and momentum equations, the homotopy perturbation method was employed. The effects of various physical parameters on temperature and velocity profiles are illustrated in Figures 2 to 9, and a comparison table is provided to validate the results. Notably, both temperature and velocity distributions increased with higher viscous dissipation. Furthermore, the velocity profile decreased with an increase in the Biot number, while the temperature profile adjacent to the plate increased as the Biot number grew. Shear stress also exhibited an upward trend with rising viscous dissipation. Finally, an increase in the Grashof number and Biot number is found to elevate the skin friction on both plates. The mean temperature is higher when air is used as the working fluid compared to mercury. To validate this study, the temperature and velocity results were compared with previously published work, showing excellent agreement. This confirms the efficiency of the Homotopy Perturbation Method in solving coupled and nonlinear system of differential equations. Additionally, it was observed that both temperature and velocity increase with a rise in the Prandtl number, attributed to the dominance of momentum diffusivity over thermal diffusivity.

## Keywords

Convective Boundary Condition, Couette Flow, Natural Convection, Viscous Dissipation, Homotopy Perturbation Method

## 1. Introduction

The study of natural convection in Couette flow has gained significant attention due to its applications in various fields, including engineering, geothermal energy, the petroleum industry, physics, nuclear power plants, and lubrication technology. Momohjimoh et al. [1] observed a reduction in velocity boundary layer thickness with increasing Biot and Prandtl numbers. Ajibade et al. [2] noted that greater internal conductive resistance diminishes buoyancy forces, resulting

in reduced fluid thickness, temperature, and velocity. Agunbiade and Oyekunle [3] reported that higher viscous dissipation enhances fluid temperature and velocity. Similarly, Tafida and Ajibade [4] highlighted the positive influence of viscous dissipation on these parameters. Ajibade and Tafida [5] found that velocity boundary layer thickness enhances with larger Biot numbers.

Astanina et al. [6] demonstrated that variable viscosity in

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the working fluid effectively controls heat transfer and fluid flow. Hussain et al. [7] confirmed that the rate of heat transfer improves with higher Prandtl numbers. Ajibade et al. [8] revealed that hydrodynamic and thermodynamic distributions improve with increased viscous dissipation, while Ajibade and Tafida [9] observed that viscous dissipation raises fluid temperature but tends to reduce velocity. Finally, Ajibade and Princely [10] concluded that greater viscous dissipation leads to higher fluid temperature and velocity within the channel. Mehdi and Kourosh [11] concluded that growing the generalized Prandtl number increases heat transfer while reducing skin friction, and also decreases the thickness of the temperature boundary layer. Muawiya and Tafida [12] observed that the mass flux increases with higher viscous dissipation (Ec). Isa et al. [13] reported that skin friction rises with an increase in the Grashof number, while the Nusselt number decreases. Hamza [14] noted that the heat source parameter increases with a rise in the Biot number. Masthanaiah et al. [15] indicated that higher values of viscous dissipation result in greater temperature generation. Tafida et al. [16] concluded that velocity and temperature distribution develop as viscous dissipation increases.

Ajibade et al. [17] observed that the velocity profile decreases adjacent to the cold plate as viscous dissipation increases. Liu et al. [18] highlighted the superiority of their method over conventional approaches, emphasizing its flexibility in selecting the initial approximation. Abker [19] successfully provided solutions for both linear and nonlinear equations by the method of homotopy perturbation (HPM). Abu-Zeid [20] found that temperature rises with an increase in the Brinkman number. Adamu [21] concluded that HPM is a robust and effective tool for solving coupled differential equations. Ajibade and Tafida [22] demonstrated the effectiveness of their HPM-based solution, achieving excellent results. Nasution [23] reported that HPM is suitable for solving nonlinear differential equation systems, as evidenced by simulations where HPM results closely matched numerical simulations using the 4th-order Runge-Kutta method. The main objective of this research is to apply the homotopy perturbation method (HPM) to examine the impact of viscous dissipation on steady natural convection Couette flow under convective boundary conditions. The governing equations for this flow are coupled and nonlinear, making them suitable for numerical schemes or approximate solution methods. Among these, the perturbation method is effective; however, its solutions are limited to small perturbation parameters. To address this limitation, the homotopy perturbation method (HPM) is introduced as an alternative. HPM offers rapid convergence, often requiring only a limited terms in the series solution to achieve high correctness. Given the nonlinearity and coupling of the governing equations, HPM is employed in this study to solve the energy and momentum equations.

## 2. Methodology

We examine a steady flow of an incompressible fluid with viscous dissipation occurring between two vertical parallel plates positioned at specific locations.  $y^* = 0$  and  $y^* = h$ , where the hot and cold walls are maintained at uniform temperatures  $T_1$  and  $T_0$ , respectively. It is assumed that the flow occur in the  $x^*$  – direction, vertically upward along the plates, while the  $y^*$  – axis is perpendicular to the plates, as depicted in Figure 1. Given the infinite length of the plates, both the velocity and temperature fields are assumed to depend solely on the spatial coordinate.  $y^*$ .

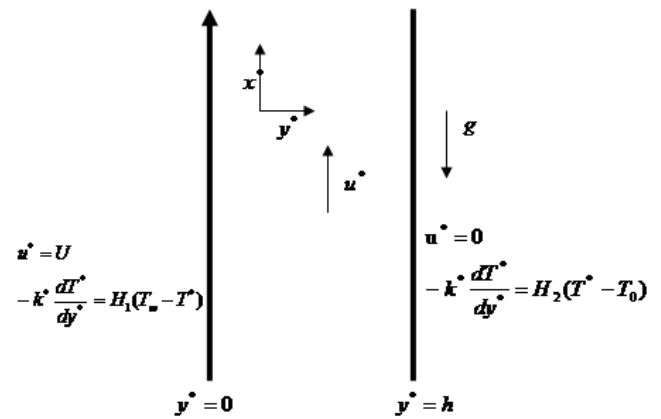


Figure 1. Schematic diagram of the problem.

The flow behavior and heat transfer in a vertical channel based on the Boussinesq's approximation, are described by the following governing equations:

$$\alpha \frac{d^2 T^*}{dy^{*2}} + \frac{\mu^*}{\rho c p} \left( \frac{du^*}{dy^*} \right)^2 = 0, \quad (1)$$

$$\nu \frac{d^2 u^*}{dy^{*2}} + g \beta (T^* - T_0) = 0. \quad (2)$$

The corresponding boundary conditions for these equations are:

$$\begin{aligned} u^* = U, \quad -k^* \frac{dT^*}{dy^*} = H_1(T_w - T^*) \quad \text{at } y^* = 0 \\ u^* = 0, \quad -k^* \frac{dT^*}{dy^*} = H_2(T^* - T_0) \quad \text{at } y^* = h. \end{aligned} \quad (3)$$

As the quantities involved have different dimensions, we introduce appropriate dimensionless variables to transform

the governing equations and boundary conditions into a dimensionless form. The dimensionless quantities used in equations (1) – (2) and boundary condition (3) are:

$$y = \frac{y^*}{h}, \quad u = \frac{u^*}{U}, \quad T = \frac{T^* - T_0}{T_w - T_0}, \quad Gr = \frac{g\beta h^2 (T_w - T_0)}{\nu^2},$$

$$Ec = \frac{U^2}{c_p (T_w - T_0)}, \quad Pr = \frac{\mu c_p}{k}, \quad Bi = \frac{hd}{k} \quad (4)$$

Using these definitions, equations (1) to (3) can be rewritten in the following non-dimensional forms:

$$\frac{d^2 T}{dy^2} + Ec Pr \left( \frac{du}{dy} \right)^2 = 0, \quad (5)$$

$$\frac{d^2 u}{dy^2} + Gr T = 0, \quad (6)$$

#### Method of Solution

To address the problem with the Homotopy Perturbation Method, we formulate a convex homotopy for the energy and momentum equations. As a result, the equations can be expressed in the following form without requiring an initial approximation:

$$\frac{d^2 T}{dy^2} = p \left[ -Ec Pr \left( \frac{du}{dy} \right)^2 \right] \quad (8)$$

such that

$$T = T_0 + pT_1 + p^2T_2 + p^3T_3 + \dots \quad (9)$$

By putting equation (9) into equation (8), we have

The non-dimensional boundary conditions are:

$$u = 1, \quad \frac{dT}{dy} = Bi(T - 1), \quad \text{at } y = 0$$

$$\frac{d^2 T_0}{dy^2} + p \frac{d^2 T_1}{dy^2} + p^2 \frac{d^2 T_2}{dy^2} + \dots = -p \left( Ec Pr \left( \frac{du_0}{dy} \right)^2 \right) - p^2 \left( 2Ec Pr \left( \frac{du_0}{dy} \cdot \frac{du_1}{dy} \right) \right) - p^3 \left( 2Ec Pr \left( \frac{du_0}{dy} \cdot \frac{du_2}{dy} - Ec Pr \left( \frac{du_1}{dy} \right)^2 \right) \right) - \dots \quad (10)$$

By relating the coefficient of  $p^0, p^1, p^2, p^3 \dots$ , we obtain

$$p^0 : \frac{d^2 T_0}{dy^2} = 0, \quad (11)$$

$$p^1 : \frac{d^2 T_1}{dy^2} = -Ec Pr \left( \frac{du_0}{dy} \right)^2, \quad (12)$$

$$p^2 : \frac{d^2 T_2}{dy^2} = -2Ec Pr \frac{du_0}{dy} \cdot \frac{du_1}{dy}, \quad (13)$$

$$p^3 : \frac{d^2 T_3}{dy^2} = -2Ec Pr \frac{du_0}{dy} \cdot \frac{du_2}{dy} - Ec Pr \left( \frac{du_1}{dy} \right)^2, \quad (14)$$

The boundary conditions are transformed to:

$$T_0'(0) = Bi(T_0 - 1), \quad T_1'(0) = BiT_1, T_2'(0) = BiT_2, T_3'(0) = BiT_3, \dots$$

$$T_0'(1) = -BiT_0, T_1'(1) = -BiT_1, T_2'(1) = -BiT_2, T_3'(1) = -BiT_3, \dots \quad (15)$$

Similarly, for equation (6), there is no initial approximation; thus, it can be transformed into:

$$\frac{d^2u}{dy^2} = p \left[ \frac{dP}{dx} - GrT \right] \quad (16)$$

such that

$$u = u_0 + pu_1 + p^2u_2 + p^3u_3 + \dots \quad (17)$$

By substituting equation (17) into equation (16), we obtain:

$$\frac{d^2u_0}{dy^2} + p \frac{d^2u_1}{dy^2} + p^2 \frac{d^2u_2}{dy^2} + \dots = -pGrT_0 - p^2GrT_1 - p^3GrT_3 - \dots \quad (18)$$

By relating the coefficient of  $p^0, p^1, p^2, p^3 \dots$ , we obtain

$$p^0 : \frac{d^2u_0}{dy^2} = 0, \quad (19)$$

$$p^1 : \frac{d^2u_1}{dy^2} = -GrT_0, \quad (20)$$

$$p^2 : \frac{d^2u_2}{dy^2} = -GrT_1, \quad (21)$$

$$p^3 : \frac{d^2u_3}{dy^2} = -GrT_2, \quad (22)$$

The boundary conditions for these equations are transformed to:

$$u_0(0) = 1, \quad u_1(0) = u_2(0) = u_3(0) = \dots = 0, \quad u_0(1) = 0, u_1(1) = 0, u_2(1) = 0, u_3(1) = 0 \dots, \quad (23)$$

Thus, by solving equations (11) and (19) and applying the boundary conditions  $T_0'(0) = Bi(T_0 - 1)$  and  $T_0'(1) = -BiT_0$ ,  $u_0(0) = 0$ , and  $u_0(1) = 0$ , we obtain the solutions

$$T_0 = A_1y + A_2 \quad (24)$$

$$u_0 = B_1y + B_2, \quad (25)$$

Thus, by solving equations (12) and (20) and applying the boundary conditions  $T_1'(0) = BiT_1$ , and  $T_1'(1) = -BiT_1$ ,  $u_1(0) = 0$ , and  $u_1(1) = 0$ , we obtain the solutions

$$T_1 = \frac{-EcPr y^2}{2} + A_3y + A_4, \quad (26)$$

$$u_1 = \frac{GrBi y^3}{6(2+Bi)} - \frac{Gr(1+Bi)y^2}{2(2+Bi)} + B_3y + B_4. \quad (27)$$

Thus, by solving equations (13) and (21) and applying the boundary conditions  $T_2'(0) = BiT_2$ , and  $T_2'(1) = -BiT_2$ ,  $u_2(0) = 0$ ,

and  $u_2(1) = 0$ , we obtain the solutions

$$T_2 = \frac{Ec Pr Gr Bi y^4}{12(2+Bi)} - \frac{Ec Pr Gr(1+Bi)y^3}{3(2+Bi)} + \frac{Ec Pr Gr(1+Bi)y^2}{2(2+Bi)} - \frac{Ec Pr Gr Bi y^2}{6(2+Bi)} + A_5 y + A_6. \quad (28)$$

$$u_2 = \frac{Ec Pr Gr y^4}{24} - \frac{A_3 Gr y^3}{6} - \frac{A_4 Gr y^2}{2} + B_5 y + B_6. \quad (29)$$

Thus, by solving equations (14) and (22) and applying the boundary conditions  $T_3'(0) = Bi T_3$ , and  $T_3'(1) = -Bi T_3$ ,  $u_3(0) = 0$ , and  $u_3(1) = 0$ , we obtain the solutions

$$\begin{aligned} T_3 = & \frac{Ec^2 Pr^2 Gr y^5}{60} - \frac{Ec Pr Gr A_3 y^4}{12} - \frac{Ec Pr Gr A_4 y^3}{6} + \frac{Ec Pr A_3 y^2}{6} + \frac{Ec Pr Gr y^2}{2} - \frac{Ec^2 Pr^2 Gr y^2}{24} - \frac{Ec Pr Gr^2 Bi^2 y^6}{120(2+Bi)^2} \\ & + \frac{Ec Pr Gr^2 Bi(1+Bi)y^5}{40(2+Bi)^2} + \frac{Ec Pr Gr^2 Bi^2 y^4}{144(2+Bi)^2} + \frac{Ec Pr Gr^2 Bi(1+Bi)y^4}{24(2+Bi)^2} - \frac{Ec Pr Gr^2(1+Bi)^2 y^3}{12(2+Bi)^2} + \frac{Ec Pr Gr^2(1+Bi)^2 y^3}{12(2+Bi)^2} \\ & - \frac{Ec Pr Gr^2 Bi(1+Bi)y^3}{36(2+Bi)^2} - \frac{Ec Pr Gr^2 Bi(1+Bi)y^3}{24(2+Bi)^2} + \frac{Ec Pr Gr^2(1+Bi)^2 y^3}{12(2+Bi)^2} - \frac{Ec Pr Gr^2(1+Bi)y^2}{8(2+Bi)^2} + \frac{Ec Pr Bi^2(1+Bi)y^2}{24(2+Bi)^2} \\ & + \frac{Ec Pr Gr^2 Bi^2 y^4}{144(2+Bi)^2} - \frac{Ec Pr Gr^2 Bi(1+Bi)y^3}{36(2+Bi)^2} + \frac{Ec Pr Gr^2 Bi(1+Bi)y^2}{24(2+Bi)^2} - \frac{Ec Pr Gr^2 Bi^2 y^2}{72(2+Bi)^2} - \frac{Ec Pr Gr^2 Bi(1+Bi)y^4}{48(2+Bi)^2} + A_7 y + A_8. \end{aligned} \quad (30)$$

$$u_3 = \frac{-Ec Pr Gr^2 y^6}{360(2+Bi)} + \frac{Ec Pr Gr^2(1+Bi)y^5}{120(2+Bi)} - \frac{Ec Pr Gr^2(1+Bi)y^4}{24(2+Bi)} + \frac{Ec Pr Gr^2 Bi y^4}{72(2+Bi)} - \frac{A_5 Gr y^3}{6} - \frac{A_6 Gr y^2}{2} + B_7 y + B_8. \quad (31)$$

Where,

$$A_1 = \frac{-Bi}{(2+Bi)}, A_2 = \frac{(1+Bi)}{(2+Bi)}, B_1 = -1, B_2 = 1,$$

$$A_3 = A_4 Bi, A_4 = \frac{1}{(2Bi+Bi^2)} \left[ \frac{Ec Pr Bi}{2} + Ec Pr \right],$$

$$B_3 = \frac{Gr(1+Bi)}{2(2+Bi)} - \frac{Gr Bi}{6(2+Bi)},$$

$$A_5 = A_6 Bi,$$

$$B_5 = \frac{A_3 Gr}{6} + \frac{A_4 Gr}{2} - \frac{Ec Pr Gr}{24},$$

$$A_6 = \frac{1}{(2Bi+Bi^2)} \left[ \frac{Ec Pr Gr Bi^2}{12(2+Bi)} - \frac{Ec Pr Gr Bi(1+Bi)}{6(2+Bi)} \right]$$

$$A_7 = A_8 Bi,$$

$$\begin{aligned}
B_7 &= \frac{Ec \, Pr \, Gr^2}{360(2+Bi)} + \frac{Ec \, Pr \, Gr^2(1+Bi)}{30(2+Bi)} - \frac{Ec \, Pr \, Gr^2 Bi}{72(2+Bi)} + \frac{A_5 Gr}{6} \\
A_8 &= \frac{1}{(2Bi+Bi^2)} \left[ \frac{Ec^2 \, Pr^2 \, Gr Bi}{40} - \frac{Ec \, Pr \, Gr A_3 Bi}{12} - \frac{Ec \, Pr \, Gr A_4 Bi}{3} + \frac{Ec \, Pr \, Gr^2 Bi^3(1+Bi)}{120(2+Bi)^2} \right] \\
&\quad \left[ -\frac{Ec \, Pr \, Gr^2 Bi(1+Bi)^2}{24(2+Bi)^2} - \frac{Ec \, Pr \, Gr Bi^3(1+Bi)}{24((2+Bi)^2)} + \frac{Ec \, Pr \, Gr A_3}{3} + \frac{Ec \, Pr \, Gr A_4}{3} \right] \\
&\quad \left[ +\frac{7Ec \, Pr \, Gr^2 Bi^2(1+Bi)}{720(2+Bi)^2} - \frac{Ec \, Pr \, Gr}{3} + \frac{Ec \, Pr \, Gr^2(1+Bi)^2}{45} - \frac{Ec \, Pr \, Gr^2(1+Bi)^2}{4} \right] \\
B_4 &= B_6 = B_8 = 0,
\end{aligned}$$

The rate of heat transfer and skin friction  $Nu_0, Nu_1, \tau_0$  and  $\tau_1$  are expressed as follows:

$$\begin{aligned}
\left. \frac{dT}{dy} \right|_{y=0} &= \frac{-Bi}{(2+Bi)} + A_3 + A_5 + A_7, \\
\left. \frac{dT}{dy} \right|_{y=1} &= \frac{-Bi}{(2+Bi)} - Ec \, Pr + A_3 + A_5 + A_7 + \frac{Ec \, Pr \, Gr A_4}{2} \\
&\quad - \frac{Ec \, Pr \, Gr^2 Bi^2}{45(2+Bi)^2} + \frac{Ec \, Pr \, Gr Bi^2(1+Bi)}{12(2+Bi)^2}, \\
\left. \frac{du}{dy} \right|_{y=0} &= -1 + \frac{Gr(1+Bi)}{2(2+Bi)} - \frac{Gr Bi}{6(2+Bi)} + \frac{A_3 Gr}{6} + \frac{A_4 Gr}{2} - \frac{Ec \, Pr \, Gr}{24} \\
&\quad + \frac{Ec \, Pr \, Gr^2}{360(2+Bi)} + \frac{Ec \, Pr \, Gr^2(1+Bi)}{30(2+Bi)} - \frac{Ec \, Pr \, Gr^2 Bi}{72(2+Bi)} + \frac{A_5 Gr}{6}, \\
\left. \frac{du}{dy} \right|_{y=1} &= -1 + \frac{Gr Bi}{3(2+Bi)} - \frac{Gr(1+Bi)}{2(2+Bi)} + \frac{Ec \, Pr \, Gr}{8} - \frac{5Gr A_4}{6} - \frac{Ec \, Pr \, Gr}{72(2+Bi)} \\
&\quad - \frac{Ec \, Pr \, Gr^2(1+Bi)}{120(2+Bi)} + \frac{Ec \, Pr \, Gr^2 Bi}{24} - \frac{A_5 Gr}{60}
\end{aligned}$$

To find the mass flux,  $Q = \int_0^1 u dy$ , we have

$$Q = \frac{1}{2} - \frac{Gr Bi}{24} + \frac{Gr(1+Bi)}{12} - \frac{Ec \, Pr \, Gr}{80} + \frac{Gr A_3}{24} + \frac{Gr A_4}{12} + \frac{Ec \, Pr \, Gr^2}{1008} + \frac{7Ec \, Pr \, Gr^2(1+Bi)}{720(2+Bi)} - \frac{Ec \, Pr \, Gr^2 Bi}{240} + \frac{Gr A_5}{36}.$$

To obtain the mean temperature,

$$Q_m = \frac{\int_0^1 u T dy}{\int_0^1 u dy}, \quad Q_m = \frac{1}{2} - \frac{GrBi}{24(2+Bi)} + \frac{Gr(1+Bi)}{12(2+Bi)} - \frac{EcPrGr^2}{2520(2+Bi)} - \frac{GrA_5}{24(2+Bi)} - \frac{Bi}{2(2+Bi)} + \frac{(1+Bi)}{(2+Bi)} - \frac{EcPr}{6} + \frac{A_3}{2} + A_4 + \frac{EcPrGrBi}{60(2+Bi)} - \frac{EcPrGr(1+Bi)}{12(2+Bi)} \quad (32)$$

### 3. Results and Discussion

This section presents a discussion of the results obtained. The analytical solutions for temperature and velocity are displayed in graphs and analyzed for various values of the monitoring parameters. Moreover, the calculated numerical values for the rate of heat transfer, skin friction, mass flux, and mean temperature are provided in tables for further evaluation.

The temperature and velocity profiles for different values of Eckert number ( $Ec$ ) is displayed in [Figures 2 and 3](#). Both distributions increase with higher Eckert numbers, as the fluid's kinetic energy surpasses the boundary layer enthalpy difference. This intensifies convection currents, reducing fluid density and resulting in higher temperature and velocity profiles within the channel. It is also noted that the effect of the Eckert number on the temperature distribution is more pronounced near the cold plate.

The effect of the thermal buoyancy ratio ( $Gr$ ) is shown in [Figures 4 and 5](#). It is observed that the temperature distribution increases near the heated plate as the Grashof number increases, while a decrease occurs near the cold plate with rising Grashof number. Additionally, the Grashof number has no noticeable effect on the temperature distribution at the center of the channel. In [Figure 5](#), the velocity distribution increases across the channel with higher  $Gr$  values, which is attributed to the buoyancy forces overpowering the viscous forces. This causes fluid particles in more energetic regions to move toward less energetic areas.

[Figures 6 and 7](#) illustrate the effects of the Prandtl number ( $Pr$ ) on the fluid's temperature and velocity distributions. It is observed that both temperature and velocity increase as the Prandtl number rises. This is due to the dominance of momentum diffusivity over thermal diffusivity. Physically, as the momentum diffusivity rate surpasses the thermal diffusivity, the fluid's boundary layer thickness increases, which enhances the convection currents within the channel, leading to higher temperature and velocity distributions.

[Figures 8 and 9](#) illustrate the effects of the Biot number ( $Bi$ ) on the velocity and temperature distributions, respectively. [Figure 8](#) shows that temperature distribution increases nearby the heated plate as the Biot number rises, while it decreases near the cold plate with an increasing Biot number. In [Figure 9](#), the velocity profile is observed to decrease as the Biot number increases. This behavior is due to the dominance of internal

conduction resistance over external convection resistance, which affects heat transfer to the fluid, consequently reducing its temperature and weakening fluid motion.

[Table 1](#) presents the computed numerical values for the rate of heat transfer at both plates. It is evident from the table that, the rate of heat transfer decreases at both plates as viscous dissipation increases. Furthermore, an increase in both the Prandtl number and viscous dissipation results in a higher rate of heat transfer.

[Table 2](#) shows the calculated numerical values of skin friction at both plates. It is observed that an increase in the Grashof number and Biot number inclines to raise the skin friction at both plates.

[Table 3](#) presents the mass flux values. Mass flux increases as the Biot number and Grashof number rise, while it decreases with an increase in the Prandtl number.

[Table 4](#) shows the mean temperature values. The mean temperature is higher when the working fluid is air compared to mercury. Additionally, the mean temperature increases with rising Grashof and Biot numbers.

#### Validation of results

To validate this study, we compared our results for temperature and velocity with those of Muawiya and Tafida [\[12\]](#), finding excellent agreement, as presented in [Table 5](#). This confirms that the Homotopy Perturbation Method is an effective approach for solving coupled and nonlinear differential equations. Additionally, to further substantiate our findings, we extended the work of Muawiya and Tafida [\[12\]](#) by introducing a different boundary condition, specifically the convective boundary condition. The comparison is outlined in [Table 5](#).

### 4. Conclusion

This article examines the effect of viscous dissipation on steady natural convection Couette flow with convective boundary conditions, employing the homotopy perturbation method. The energy and momentum equations were solved, and the results were presented graphically and analyzed for various values of the governing parameters. The study concludes that:

Both temperature and velocity profiles rise with increasing viscous dissipation,

The velocity profile decreases as the Biot number increases,

A rise in the Prandtl number and viscous dissipation leads to an enhanced rate of heat transfer,

The mean temperature is greater when air is used as the working fluid compared to mercury.

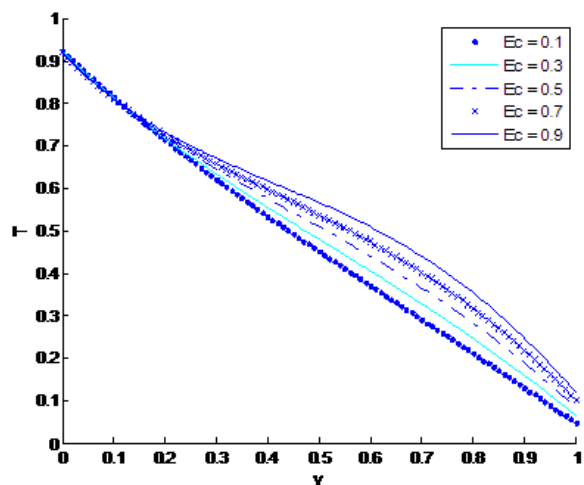


Figure 2. Temperature profile for various values of  $Ec$  ( $Pr = 0.71, Gr = 6.0, Bi = 15.0$ ).

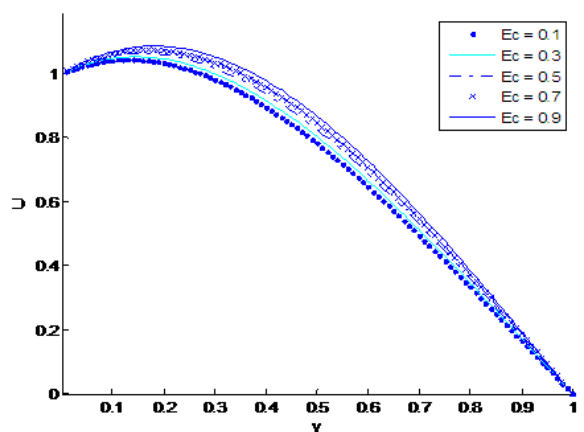


Figure 3. Velocity profile for various values of  $Ec$  ( $Pr = 0.71, Gr = 6.0, Bi = 15.0$ ).

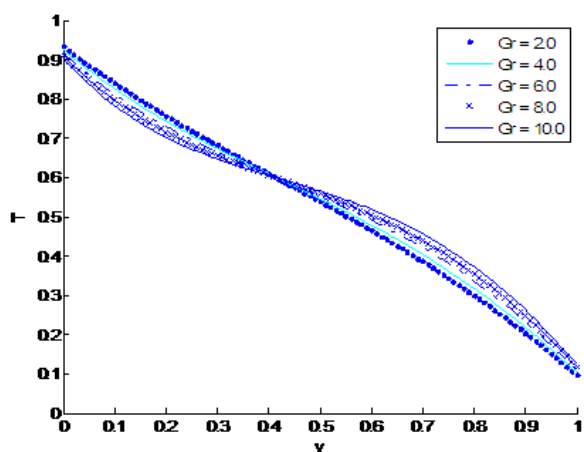


Figure 4. Temperature profile for various values of  $Gr$  ( $Ec = 0.8, Pr = 0.71, Bi = 15.0$ ).

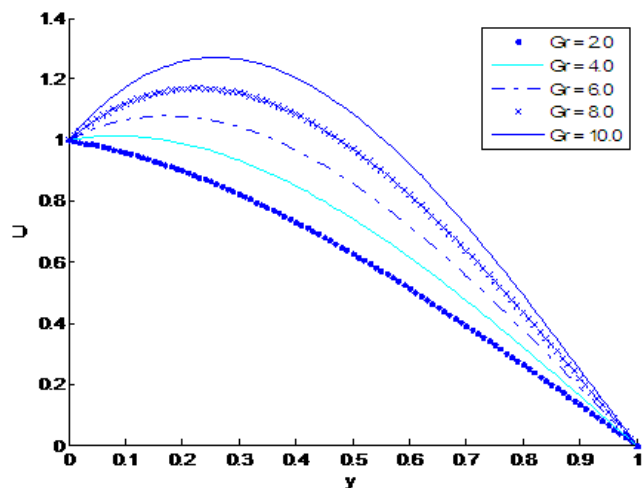


Figure 5. Velocity profile for various values of  $Gr$  ( $Ec = 0.8, Pr = 0.71, Bi = 15.0$ ).

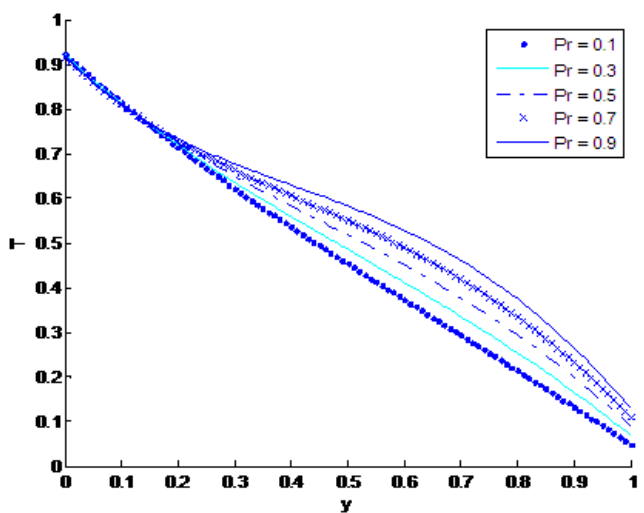


Figure 6. Velocity profile for various values of  $Pr$  ( $Ec = 0.8, Gr = 6.0, Bi = 15.0$ ).

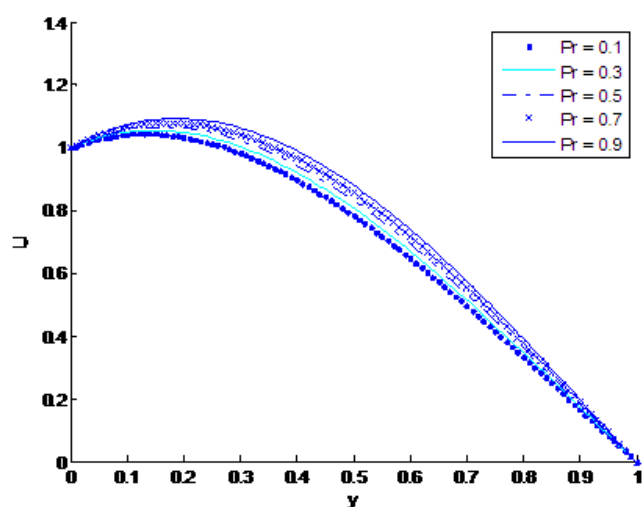
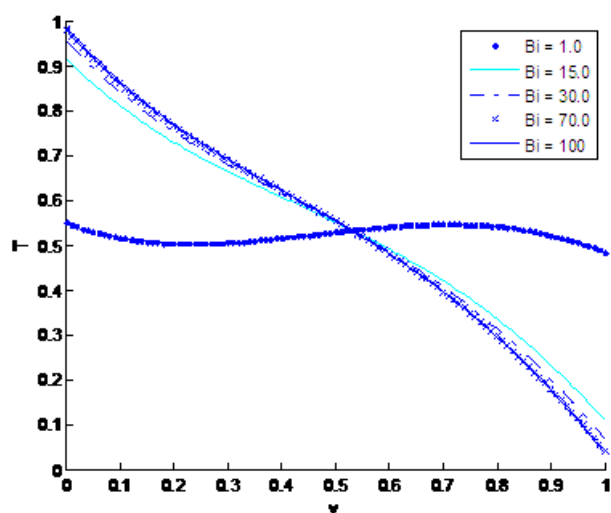
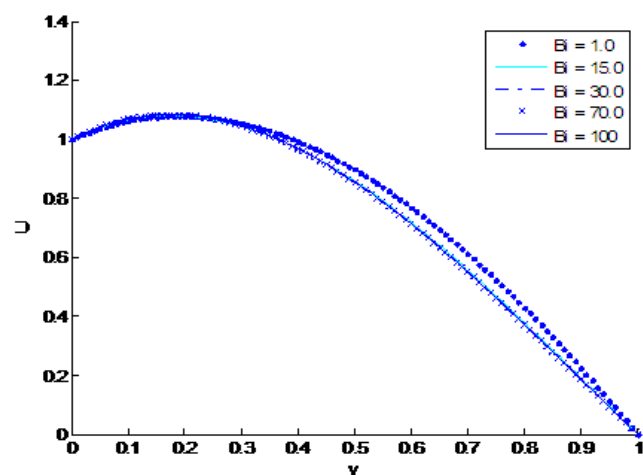


Figure 7. Temperature profile for various values of  $Pr$  ( $Ec = 0.8, Gr = 6.0, Bi = 15.0$ ).





**Figure 8.** Velocity profile for various values of  $Bi$  ( $Ec = 0.8, Pr = 0.71, Gr = 6.0$ ).



**Figure 9.** Temperature profile for various values of  $Bi$  ( $Ec = 0.8, Pr = 0.71, Gr = 6.0$ ).

**Table 1.** Calculated numerical values for the rate of heat transfer at both plates.

$Ec$	$Pr$	$Gr = 6.0, Bi = 7.0$		$Gr = 8.0, Bi = 7.0$		$Gr = 10.0, Bi = 17.0$	
		$Nu_1$	$Nu_0$	$Nu_0$	$Nu_1$	$Nu_0$	$Nu_1$
0.1	0.044	0.6821	0.4346	0.6778	0.2668	0.4821	0.3353
0.4		0.3631	0.2538	0.3517	0.1287	0.3007	0.2078
0.7		0.2741	0.1780	0.2130	0.1007	0.2706	0.2006
1.0		0.0761	0.0431	0.0357	0.0927	0.1908	0.0990
0.1	0.71	0.5793	0.6110	0.4468	0.3700	0.4011	1.4462
0.4		0.2411	0.4240	0.1311	0.2356	0.2968	1.3207
0.7		0.1630	0.2086	0.1827	0.1971	0.2077	1.2006
1.0		0.0835	0.1781	0.1600	0.1736	0.1243	1.1371

**Table 2.** Calculated numerical values for the skin friction at both plates.

$Ec$	$Pr$	$Gr = 6.0, Bi = 7.0$		$Gr = 8.0, Bi = 7.0$		$Gr = 10.0, Bi = 17.0$	
		$\tau_1$	$\tau_0$	$\tau_0$	$\tau_1$	$\tau_0$	$\tau_1$
0.1	0.044	0.5261	1.6732	0.5273	1.6821	1.3281	1.5281
0.4		0.5700	1.7260	0.5738	1.7386	1.5711	1.6525
0.7		0.6134	1.9980	0.6380	1.9990	1.7013	1.9752
1.0		0.6531	2.3600	0.6597	2.4987	1.8245	2.3655
0.1	0.71	0.4217	1.4894	0.4256	1.4910	0.8525	1.0992

$Ec$	$Pr$	$Gr = 6.0, Bi = 7.0$		$Gr = 8.0, Bi = 7.0$		$Gr = 10.0, Bi = 17.0$	
		$\tau_1$	$\tau_0$	$\tau_0$	$\tau_1$	$\tau_0$	$\tau_1$
0.4	0.71	0.4530	1.5360	0.4620	1.5470	1.0710	1.3553
0.7		0.4986	1.8671	0.5211	1.8710	1.5051	1.8520
1.0		0.5246	1.9876	0.5632	1.9940	1.5652	2.2505

**Table 3.** Calculated numerical values of mass flux  $Q$ .

$Ec$	$Pr$	$Gr = 6.0, Bi = 7.0$	$Gr = 8.0, Bi = 7.0$	$Gr = 8.0, Bi = 7.0$
		$Q$	$Q$	$Q$
0.1	0.044	0.7685	0.7692	0.8235
0.4		0.7812	0.7850	0.8857
0.7		0.8150	0.8171	0.8891
1.0		0.8301	0.8380	0.9521
0.1	0.71	0.7445	0.7451	0.7425
0.4		0.7581	0.7598	0.7901
0.7		0.7805	0.7820	0.8873
1.0		0.8133	0.8182	0.849

**Table 4.** Calculated numerical values of mean temperature  $Q_m$ .

$Ec$	$Pr$	$Gr = 6.0, Bi = 7.0$	$Gr = 8.0, Bi = 7.0$	$Gr = 8.0, Bi = 7.0$
		$Q_m$	$Q_m$	$Q_m$
0.1	0.044	8.7552	8.7926	8.9572
0.4		4.0361	4.0675	4.2531
0.7		3.8501	3.9172	3.9721
1.0		5.8424	5.9715	6.2140
0.1	0.71	10.2706	11.6981	12.2531
0.4		5.2961	6.3284	7.9602
0.7		4.5072	5.7561	6.4821
1.0		6.8721	7.8815	8.1236

**Table 5.** Comparison of calculated numerical values between the present problem and that of Muawiya and Tafida [12].

Muawiya and Tafida [12]			Present problem	
$Gr = 6.0, Pr = 0.71, y = 0.5$			$Gr = 6.0, Pr = 0.71, Bi = 130, y = 0.5$	
Ec	Temperature	Velocity	Temperature	Velocity
0.1	0.4824	0.7761	0.4817	0.7703
0.4	0.4607	0.7538	0.4601	0.7510
0.7	0.4210	0.7210	0.4178	0.7193
1.0	0.3986	0.6993	0.3952	0.6907

## Abbreviations

$g$	Acceleration to Due Gravity [ $\text{ms}^{-2}$ ]
$\beta$	Coefficient of Thermal Expansion [ $\text{K}^{-1}$ ]
$h$	Width of the Channel [m]
$\mu$	Coefficient of Viscosity [ $\text{Kgm}^{-1}\text{s}^{-1}$ ]
$T^*$	Dimensional Fluid Temperature [K]
$\nu$	Kinematic Viscosity [ $\text{m}^2\text{s}^{-1}$ ]
$T_w$	Channel Wall Temperature [K]
$Pr$	Prandtl Number
$T_0$	Temperature of the Ambience [K]
$Gr$	Grashof Number
$T$	Dimensionless Fluid Temperature
$Ec$	Eckert Number
$u^*$	Dimensional Velocity [ $\text{ms}^{-1}$ ]
$Bi$	Biot Number
$u$	Dimensionless Velocity
$c_p$	Specific Heat at Constant Pressure [ $\text{m}^2\text{s}^{-2}\text{K}^{-1}$ ]
$U$	Dimensional Velocity of the Moving Plate [ $\text{ms}^{-1}$ ]
$\rho$	Density of the Fluid [ $\text{Kgm}^{-3}$ ]
$y^*$	Coordinate Perpendicular to the Plate [m]
$\alpha$	Thermal Diffusivity of the Fluid [ $\text{Kgm}^{-3}$ ]
$y$	Dimensionless Co-ordinate Perpendicular to the Plate

## Conflicts of Interest

The authors declare no conflicts of interest.

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